Attitude Stabilization Using Modified Rodrigues Parameters without Angular Velocity Measurements

Awad El-Gohary, Tawfik El-Sayed Tawfik

Department of Mathematics, Faculty of Science, Mansoura University, Mansoura, Egypt E-mail: aigohary@ksu.edu.sa, tawfikstm@yahoo.com Received January 26, 2011; revised April 6, 2011; accepted April 7, 2011

Abstract

The optimal stabilization of a rigid body motion without angular velocity measurements is considered with the help of three internal rotors that effected by internal frictions. In this paper, the orientation of the body will be described in terms of the Modified Rodrigues parameters (MRPs). The optimal control law which stabilizes asymptotically this motion and minimizes the require like-energy cost is obtained in terms of the MRPs. Numerical study and simulation are introduced.

Keywords: Attitude Control, Rotors with Frictions, Modified Rodrigues Parameters

1. Introduction

Most research into attitude motions of rigid bodies systems always has been and still remains one of the important problems of theoretical and applied mechanics. The controlling of a rigid body motion means how we can select the control law that ensures an asymptotic stability of this motion. Physically the control transfers the state of rigid body from an arbitrary initial state to the desired state. This control law is considered to be optimal if it minimizes a selecting performance index. This problem is considered one of the important problem in modern mechanics since the rigid body is a suitable mathematical and physical model for study the motion of satellite, aircraft, spacecraft and the like.

Many studies have derived the control laws in terms of the angular velocities of the rigid body and parameters attitude the orientation of the rigid body with respect to the inertial axes (El-Gohary, 2005a; Izzo and Pettazzi, 2007; Junfengy *et al.*, 2000; Lovera and Astolfib, 2004; Tayebi and McGilvray, 2006; Tsiotras *et al.*, 2001). The angular velocity measurement is noisy, that is it contains high-frequency and noises or random fluctuations. The controlling of a rigid body motion without angular velocity measurement using control torques is studied in (Akella, 2001; Costic *et al.*, 2000; Lizarralde and Wen, 1996; Tayebi, 2006; Tsiotras, 1995). El-Gohary and Tawfik (2010) studied the optimal stabilization of a rotational motion of a rigid body using three rotors with internal friction moments. The control law which stabilizes asymptotically this motion is obtained in terms of Euler parameters. El-Gohary (2005b) studied the optimal stabilization of an equilibrium position using three rotors without inertial frictions. The control law which stabilizes asymptotically this position is obtained either in terms of the Cayley-Rodrigues parameters, or in terms of the MRPs. In this paper, the control law which stabilizes asymptotically a rotational motion of a rigid body in terms of the MRPs is derived. We will take into account the inertial frictions of the rotors. Moreover, a special case of the studied problem is obtained.

2. Equations of Motion

We consider the rotational motion of a rigid body carrying three symmetrical rotors attached to the principal axes of inertia of the body. The rotational motion of a rigid body about its centre of mass is described by the following equation:

$$A\dot{\boldsymbol{\omega}} = A\boldsymbol{\omega} \wedge \boldsymbol{\omega} + I\boldsymbol{\phi} \wedge \boldsymbol{\omega} - I\boldsymbol{\phi}, \qquad (1)$$

where $\boldsymbol{\omega} = (\omega_1, \omega_2, \omega_3)^T$, $\boldsymbol{\phi} = (\phi_1, \phi_2, \phi_3)^T$, denote the angular velocity vector of the body the rotor angles of rotation vector referred to the principal axes of inertia of the body, *A* and *I* are the inertia matrices of the body and the rotors, respectively.

The equation of the relative motion of the rotors with the inertial friction moments is:

$$I\left(\dot{\boldsymbol{\omega}}+\ddot{\boldsymbol{\phi}}\right)=\boldsymbol{u}-C\dot{\boldsymbol{\phi}},\qquad(2)$$



where $\boldsymbol{u} = (u_1, u_2, u_3)^T$ denote the control vector applied to the rotors and created by electric motors rigidly mounted on the body and $C = \text{diag}(C_1, C_2, C_3)$ is diagonal friction coefficients matrix. The terms $C\phi$ due to the rotors friction. Note that the friction coefficients depend upon various factors such as temperature, angular velocity and other factors. This friction is quasi-viscous.

In this paper, the orientation of the body is described by using the MRPs. The kinematic equation in terms of the MRPs takes the form (Schaub and Junkins, 1996):

$$4\dot{\boldsymbol{\mu}} = \chi(\boldsymbol{\mu})\boldsymbol{\omega} \tag{3}$$

where

$$\psi(\boldsymbol{\mu}) = (1 + \boldsymbol{\mu}^{T} \boldsymbol{\mu})^{-2} \begin{bmatrix} 4(\mu_{1}^{2} - \mu_{2}^{2} - \mu_{3}^{2}) + \Sigma^{2} \\ 8\mu_{2}\mu_{1} - 4\mu_{3}\Sigma \\ 8\mu_{3}\mu_{1} + 4\mu_{2}\Sigma \end{bmatrix}$$

where

$$\Sigma = 1 - \boldsymbol{\mu}^{\mathrm{T}} \boldsymbol{\mu}.$$

Modified Rodrigues parameters are three parameters which have the advantage over the Cayley-Rodrigues parameters that allow eigenaxis rotations greater than 180° but can't be used to describe eigenaxis rotations of more than 360°.

The system admits the first integrals

$$\psi(\boldsymbol{\mu})(\boldsymbol{A}\boldsymbol{\omega}+\boldsymbol{I}\boldsymbol{\phi}) = \boldsymbol{h} \tag{7}$$

where $\mathbf{h} = (h_1, h_2, h_3)^{\mathrm{T}}$ is the angular momentum vector of the whole system referring to the inertial axes can be regarded as constant.

Solving Equation (7) with respect to $(A\boldsymbol{\omega} + I\dot{\boldsymbol{\phi}})$ we obtain:

$$\left(A\boldsymbol{\omega}+I\dot{\boldsymbol{\phi}}\right)=\boldsymbol{\psi}^{\mathrm{T}}\left(\boldsymbol{\mu}\right)\boldsymbol{h}.$$
(8)

The vectors $\dot{\phi}$ and $\ddot{\phi}$ can be eliminated from Equations (1) by using Equations (8) and (2), we get

$$(A-I)\dot{\boldsymbol{\omega}} = \left[\boldsymbol{\upsilon} + S^{\mathrm{T}}(\boldsymbol{\omega})\right]\boldsymbol{\psi}^{\mathrm{T}}(\boldsymbol{\mu})\boldsymbol{h} - \boldsymbol{\upsilon}\boldsymbol{A}\boldsymbol{\omega} - \boldsymbol{\mu} \quad (9)$$

$$\chi(\boldsymbol{\mu}) = \left(1 - \boldsymbol{\mu}^{\mathrm{T}} \boldsymbol{\mu}\right) I_{3\times 3}^{*} + 2S(\boldsymbol{\mu}) + 2\boldsymbol{\mu} \boldsymbol{\mu}^{\mathrm{T}} \qquad (4)$$

 I^* is the 3×3 unit matrix. The matrix $S(\mu)$ denotes the following skew symmetric matrix:

$$S(\boldsymbol{\mu}) = \begin{bmatrix} 0 & -\mu_3 & \mu_2 \\ \mu_3 & 0 & -\mu_1 \\ -\mu_2 & \mu_1 & 0 \end{bmatrix}.$$
 (5)

The direction cosines matrix of the inertial axes relative to the principal axes of inertia of the body in terms of the MRPs can be written in the form (Schaub and Junkins, 1996):

$$\begin{array}{cccc}
8\mu_{1}\mu_{2} + 4\mu_{3}\Sigma & 8\mu_{1}\mu_{3} - 4\mu_{2}\Sigma \\
4\left(\mu_{2}^{2} - \mu_{1}^{2} - \mu_{3}^{2}\right) + \Sigma^{2} & 8\mu_{2}\mu_{3} + 4\mu_{1}\Sigma \\
8\mu_{3}\mu_{2} - 4\mu_{1}\Sigma & 4\left(\mu_{3}^{2} - \mu_{1}^{2} - \mu_{2}^{2}\right) + \Sigma^{2}
\end{array}$$
(6)

where $v = \text{diag}(C_1/I_1, C_2/I_2, C_3/I_3)$ is positive diagonal matrix.

The system (9) and (3) admit the special solution

$$\boldsymbol{\omega}^{(r)} = \boldsymbol{\omega}\boldsymbol{e}, \ \boldsymbol{\mu}^{(r)} = \boldsymbol{e} \tan\left(\boldsymbol{\omega}t/4\right)$$
(10)

if the control vector take the values

$$\boldsymbol{u}^{(r)} = \left[\boldsymbol{\upsilon} + \boldsymbol{\omega} S^{\mathrm{T}}\left(\boldsymbol{e}\right)\right] \boldsymbol{\psi}^{\mathrm{T}}\left(\boldsymbol{\mu}^{(r)}\right) \boldsymbol{h} - \boldsymbol{\omega} \boldsymbol{\upsilon} \boldsymbol{A} \boldsymbol{e} \qquad (11)$$

where $\boldsymbol{e} = (e_1, e_2, e)^{T}$ is the unit vector that the body rotates about it, referred to the principal axes of inertia of the body. The solution (10) represents a rotational motion of the body around an axis which is fixed in the body with a certain angular velocity ω .

The optimal control law applied to the rotors which stabilizes asymptotically the rotational motion (10) of the rigid body can be determined only in terms of the MRPs and their estimates without angular velocity measurements. To derive these control law, we start by introducing the new variables which represent estimates attitude parameters. Assume that $\hat{\mu}$ is an estimate of the kinematic attitude vector μ . Also we suppose that the kinematic attitude vector and its estimate satisfy the following auxiliary system of differential equations:

$$E(\boldsymbol{\mu}-\hat{\boldsymbol{\mu}})\dot{\hat{\boldsymbol{\mu}}} = LE(\boldsymbol{\mu}-\hat{\boldsymbol{\mu}})(\boldsymbol{\mu}-\hat{\boldsymbol{\mu}}) + \omega E(\boldsymbol{\mu}-\hat{\boldsymbol{\mu}})\chi(\boldsymbol{\mu})\boldsymbol{e}/4 + \omega E(\boldsymbol{\mu}-\boldsymbol{\mu}^{(r)})\chi(\boldsymbol{\mu}-\boldsymbol{\mu}^{(r)})\boldsymbol{e}/4$$
(12)

where $E(\mu - \hat{\mu}) = \text{diag}(\mu_1 - \hat{\mu}_1, \mu_2 - \hat{\mu}_2, \mu_3 - \hat{\mu}_3)$ $E(\mu - \hat{\mu}) = \text{diag}(\mu_1 - \hat{\mu}_1^{(r)}, \mu_2 - \hat{\mu}_2^{(r)}, \mu_3 - \hat{\mu}_3^{(r)})$ and $L = \text{diag}(l_1, l_2, l_3)$ is positive diagonal matrix l_i (i = 1, 2, 3)

This system is known as the auxiliary system. Using the kinematic Equation (3) the auxiliary system (12) can be written in the form:

are called the stability constants.

$$E(\boldsymbol{\xi})\dot{\boldsymbol{\xi}} = -LE(\boldsymbol{\xi})\boldsymbol{\xi} + \omega E(\boldsymbol{\xi})\boldsymbol{\chi}(\boldsymbol{\mu})(\boldsymbol{\omega} - \omega \boldsymbol{e})/4 - \omega E(\boldsymbol{\mu} - \boldsymbol{\mu}^{(r)})\boldsymbol{\chi}(\boldsymbol{\mu} - \boldsymbol{\mu}^{(r)})\boldsymbol{e}/4$$
(13)

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where

$$\boldsymbol{\xi} = \boldsymbol{\mu} - \hat{\boldsymbol{\mu}} \ . \tag{14}$$

The problem is equivalent to find the optimal control law such that the rotational motion

$$\boldsymbol{\omega}^{(r)} = \boldsymbol{\omega}\boldsymbol{e}, \, \boldsymbol{\mu}^{(r)} = \boldsymbol{e} \tan\left(\boldsymbol{\omega}t/4\right), \, \boldsymbol{\xi} = (0, 0, 0)^{\mathrm{T}}, \\ \boldsymbol{u}^{(r)} = \left[\boldsymbol{\upsilon} + \boldsymbol{\omega}\boldsymbol{S}^{\mathrm{T}}\left(\boldsymbol{e}\right)\right]\boldsymbol{\psi}^{\mathrm{T}}\left(\boldsymbol{\hat{\mu}}^{(r)}\right)\boldsymbol{h} - \boldsymbol{\omega}\boldsymbol{\upsilon}\boldsymbol{A}\boldsymbol{e} \right\}$$
(15)

is asymptotically stable and minimize a selected performance.

$$(A-I)\dot{W} = \left[\upsilon + \omega S^{\mathrm{T}}(\boldsymbol{e})\right] \left[\psi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right) - \psi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\lambda}^{(r)}\right) - \psi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right) W + \omega \chi(\boldsymbol{\delta})\boldsymbol{e}, \qquad (18)$$

$$\boldsymbol{E}(\varsigma)\dot{\varsigma} = -\boldsymbol{L}\boldsymbol{E}(\varsigma)\varsigma + \omega\boldsymbol{E}(\varsigma)\boldsymbol{\chi}(\varsigma + \boldsymbol{\mu}^{(r)})\boldsymbol{W}/4 \qquad (19)$$

$$-\omega E(\delta) \chi(\delta) e/4$$

if $U_i = 0$ (i = 1, 2, 3) then the systems (17), (18) and (19) have an obvious unstable solution

$$W_i = 0, \, \delta_i = 0, \, \varsigma_i = 0, \, (i = 1, 2, 3),$$
 (20)

which must be stabilized. This stabilization can be achieved by using the control moments U_i (i = 1, 2, 3).

3. Optimal Stabilization Problem

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In this section, the optimal control law U which stabi-

To obtain the equations of the perturbed motion about

$$\boldsymbol{W} = \boldsymbol{\omega} - \boldsymbol{\omega}\boldsymbol{e}, \, \boldsymbol{\delta} = \boldsymbol{\mu} - \boldsymbol{\mu}^{(r)}, \, \boldsymbol{\varsigma} = \boldsymbol{\xi}, \, \boldsymbol{U} = \boldsymbol{u} - \boldsymbol{u}^{(r)} \quad (16)$$

where $W_i, \delta_i, \varsigma_i$ and U_i (*i* = 1, 2, 3) are the perturbation of the angular velocity of the body, MRPs, error attitude parameters and control moments about the rotational state (16) at respectively.

Substituting from (16) into (9), (3) and (13) we get the following systems:

$$= \left[\upsilon + \omega S^{\mathrm{T}}(\boldsymbol{e})\right] \left[\psi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right) - \psi^{\mathrm{T}}\left(\boldsymbol{\mu}^{(r)}\right)\right] \boldsymbol{h} + S^{\mathrm{T}}(\boldsymbol{W})\psi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right) \boldsymbol{h} - \upsilon A \boldsymbol{W} - \boldsymbol{U},$$
(17)

lizes asymptotically the zero solution (20) and minimizes an integral performance index is determined on terms of the MRPs. The asymptotic stability of the zero solution (20) is derived. Moreover, a special case of the studied problem is obtained.

Theorem. The optimal control law

$$\boldsymbol{U}^{(0)} = \left[\boldsymbol{\upsilon} + \boldsymbol{\omega}\boldsymbol{S}^{\mathrm{T}}\left(\boldsymbol{e}\right)\right] \left[\boldsymbol{\psi}^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right) - \boldsymbol{\psi}^{\mathrm{T}}\left(\boldsymbol{\mu}^{(r)}\right)\right]\boldsymbol{h} + k\boldsymbol{\chi}^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right)\left(\boldsymbol{\delta} + \boldsymbol{\varsigma}\right)/4$$
(21)

stabilizes asymptotically the zero solution (20) of the system described by Equations (17), (18) and (19) and minimizes the integral performance index:

 $2\Phi = \boldsymbol{W}^{\mathrm{T}} \left(\boldsymbol{A} - \boldsymbol{I} \right) \boldsymbol{W} + \boldsymbol{k} \left[\boldsymbol{\delta}^{\mathrm{T}} \boldsymbol{\delta} + \boldsymbol{\varsigma}^{\mathrm{T}} \boldsymbol{\varsigma} \right]$

where (A-I) is positive diagonal matrix. This func-

tion is a positive definite with respect to stabilize vari-

ables since it consists of the sum of quadratic terms. Us-

ing the Krasovskii's theorem (Krasovskii, 1966), we have

$$I = \int_{0}^{\infty} \Omega(\boldsymbol{W}, \boldsymbol{\delta}, \boldsymbol{\varsigma}, \boldsymbol{U}, t) dt$$

$$= \int_{0}^{\infty} \left\{ \boldsymbol{W}^{\mathrm{T}} \boldsymbol{\upsilon} A \boldsymbol{W} + k \boldsymbol{\delta}^{\mathrm{T}} L \boldsymbol{\delta} + \left\| \left[\boldsymbol{\upsilon} + \boldsymbol{\omega} S^{\mathrm{T}} \left(\boldsymbol{e} \right) \right] \left[\boldsymbol{\psi}^{\mathrm{T}} \left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)} \right) - \boldsymbol{\psi}^{\mathrm{T}} \left(\boldsymbol{\mu}^{(r)} \right) \right] \boldsymbol{h} + k \boldsymbol{\chi}^{\mathrm{T}} \left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)} \right) (\boldsymbol{\delta} + \boldsymbol{\varsigma}) / 4 - \boldsymbol{U} \right\|^{2} \right\} dt \right\}$$

$$(22)$$

where k is positive control constant. It should be clear that the structure of optimal feedback control law depends upon the choice of the integral performance index.

Proof: Assume that, the optimal Liapunov function in the form

$$B[\Phi, \boldsymbol{W}, \boldsymbol{\delta}, \boldsymbol{\varsigma}, \boldsymbol{U}, t] = \frac{\partial \Phi}{\partial t} + (\nabla_{\boldsymbol{W}} \Phi) \dot{\boldsymbol{W}} + (\nabla_{\boldsymbol{\delta}} \Phi) \dot{\boldsymbol{\delta}} + (\nabla_{\boldsymbol{\varsigma}} \Phi) \dot{\boldsymbol{\varsigma}} + \Omega(\boldsymbol{W}, \boldsymbol{\delta}, \boldsymbol{\varsigma}, \boldsymbol{U}, t) \ge 0.$$
(24)

Using the conditions of optimality and assuming that $\boldsymbol{U}^{(0)} = \left(U_1^{(0)}, U_2^{(0)}, U_3^{(0)} \right)^{\mathrm{T}}$ is the optimal control vector which stabilizes asymptotically the zero solution (20), the function Φ must satisfy the following partial differential equation:

$$\frac{\partial \Phi}{\partial t} (\nabla_{W} \Phi) \dot{W} + (\nabla_{\delta} \Phi) \dot{\delta} + (\nabla_{\varsigma} \Phi) \dot{\varsigma} + \Omega = 0.$$
 (25)

Substituting the function (17), (18), (19), (22) and (23) into the partial differential Equation (25), we get

$$W^{\mathrm{T}}\left\{\left[\upsilon + \omega S^{\mathrm{T}}(\boldsymbol{e})\right]\left[\psi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right) - \psi^{\mathrm{T}}\left(\boldsymbol{\mu}^{(r)}\right)\right]\boldsymbol{h} + k\chi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right)(\boldsymbol{\delta} + \boldsymbol{\varsigma})/4 - \boldsymbol{U}\right\} + \left\|\left[\upsilon + \omega S^{\mathrm{T}}(\boldsymbol{e})\right]\left[\psi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right) - \psi^{\mathrm{T}}\left(\boldsymbol{\mu}^{(r)}\right)\right]\boldsymbol{h} + k\chi^{\mathrm{T}}\left(\boldsymbol{\delta} + \boldsymbol{\mu}^{(r)}\right)(\boldsymbol{\delta} + \boldsymbol{\varsigma})/4 - \boldsymbol{U}\right\|^{2} = 0.\right\}$$
(26)

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(23)

(25)

Thus, the optimal control vector $U^{(0)}$ satisfies the Equation (26) are given by (21). Obviously, the present control law depend upon the kinematic attitude parameters and the friction coefficients. Moreover this control law do not require a knowledge of the rigid body inertial moments.

Now we will prove that the zero solution (20) is asymptotically stable under the control law (21). The total time derivative of the optimal Liapunov function (23) using (17), (18), (19) and taking into consideration the optimal control law (21) takes the form

$$\frac{d\Phi}{dt} = -\left[\boldsymbol{W}^{\mathrm{T}}\boldsymbol{\upsilon}\boldsymbol{A}\boldsymbol{W} + k\boldsymbol{\varsigma}^{\mathrm{T}}\boldsymbol{L}\boldsymbol{\varsigma}\right] \leq 0.$$
(27)

The function (23) is a positive definite with respect to the angular velocities of the body, the kinematic attitude parameters and the error attitude parameters. Furthermore, the total derivative of this function as given by (27) is a negative semi-definite function (constant sign function) only. Thus, under the optimal control law (21), the zero solution (20) is only stable in the Liapunov sense, but not necessarily asymptotic stable.

To prove the asymptotic stability of the zero solution (20) we consider the following function:

$$\Phi_{1} = 4W^{\mathrm{T}} (A - I) \eta,$$

$$\eta = -\chi \left(\delta + \mu^{(r)} \right) \delta.$$
(28)

Using Equations (17) and (21) on the set ($\dot{\Phi} = 0$, $W_1 = W_2 = W_3 = 0$, $\zeta_1 = \zeta_2 = \zeta_3 = 0$) the derivative of the function (28) is given by

$$\dot{\boldsymbol{\Phi}}_1 = k \left(\boldsymbol{\eta}^{\mathrm{T}} \boldsymbol{\eta} \right). \tag{29}$$

The function $\dot{\Phi}_1$ is a positive definite function. Thus by using theorem (1.2) in (Matrosov, 1962), the rotational motion (15) is asymptotically stable in the Liapunov sense.

Similarly when neglected the friction of the rotors system *i.e.* \boldsymbol{v} is zero matrix. Thus, $\dot{\Phi} = 0$ only if $\zeta_1 = \zeta_2 = \zeta_3 = 0$ and $W_i, \delta_i (i = 1, 2, 3)$ begin arbitrary. But in this case one can find that $\hat{\delta}_i = \delta_i (i = 1, 2, 3)$ which leads to the kinematic attitude parameters $\delta_i (i = 1, 2, 3)$ are constants. In this case the Equations (18) and (19) reduce to $W_i = 0, (i = 1, 2, 3)$. Thus, $\delta_i = 0, (i = 1, 2, 3)$.

From the above analysis, we conclude that $\dot{\Phi} = 0$ if and only if $W_i = 0$, $\varsigma_i = 0$, $\delta_i = 0$ (i = 1, 2, 3) Thus the zero solution (20) is asymptotically stable in the Liapunov sense. \Box

The equilibrium position of the rigid body which occurs when the principal axes of inertia of the body coincide with the inertial axes can be obtained as a special case of the studied problem by setting $\omega = 0$ in Equation (15):

$$\omega_i = 0, \, \mu_i = 0, \, \xi_i = 0, \, u_i = v_i h_i, \, (i = 1, 2, 3)$$
(30)

The optimal control law which stabilizes asymptotically this position can be obtained from Equation (21) by setting $\omega = 0$ and $\mu^{(r)} = (0,0,0)^{T}$

$$\boldsymbol{U}_{1}^{(E)} = \boldsymbol{\upsilon} \Big[\boldsymbol{\psi}^{\mathrm{T}} \left(\boldsymbol{\delta} \right) - \boldsymbol{I}_{3\times 3}^{*} \Big) \Big] \boldsymbol{h} + k \boldsymbol{\chi}^{\mathrm{T}} \left(\boldsymbol{\delta} \right) \left(\boldsymbol{\delta} + \boldsymbol{\varsigma} \right) / 4 . (31)$$

When the rotors system move without friction, that is, the coefficients of the friction $C_1 = C_2 = C_3 = 0 \Rightarrow v$ is zero matrix, the equilibrium position (30) reduce to

$$\omega_i = 0, \, \mu_i = 0, \, \xi_i = 0, \, u_i = 0, \, (i = 1, 2, 3).$$
 (32)

The optimal control law which stabilizes asymptotically this position can be obtained from Equation (31) by setting v zero matrix

$$\boldsymbol{U}^{(E)} = k \boldsymbol{\chi}^{\mathrm{T}} (\boldsymbol{\delta}) (\boldsymbol{\delta} + \boldsymbol{\varsigma}) / 4.$$
 (33)

This result, as a special case of the obtained results for the considered problem, agrees with the result deduced by (El-Gohary, 2005b). This shows that the present method is more general than the method used by (El-Gohary, 2005b).

We compare the optimal control law (21) and the previous control law publication on this topic (Akella, 2001; Costic *et al.*, 2000; Lizarralde and Wen, 1996; Tayebi, 2006; Tsiotras, 1995). The control law (21) stabilizes asymptotically the general rotational motion of a rigid body not an equilibrium position. Moreover, this control law applied to the internal rotors that effected by frictions not control torques.

4. Numerical Examples

The results of this paper are more useful for the rigid body application such as satellite, spacecraft, aircraft and others. In this section, we study the effect of the control constants and stability constants of the angular velocity, MRPs, error attitude parameters and control moments. We adopt the numerical values of a rigid body rotating around the third axis of inertia of the body with an angular velocity $\omega = 0.2 \text{ rad/s}$, the inertial moments of a rigid body A_1, A_2, A_3 , the inertial moments of the stabilizer rotors system I_1, I_2, I_3 , the constants of the angular momentum of the system h_1, h_2, h_3 , initial values of the angular velocities of the rigid body $W_1(t), W_2(t), W_3(t)$, MRPs $\mu_i(t)$ and their estimates $\hat{\mu}_i(t)$ as follows:

$$A_1 = 10 \text{ kgm}^2$$
, $A_2 = 15 \text{ kgm}^2$, $A_3 = 20 \text{ kgm}^2$,

$$I_1 = 0.8 \text{ kgm}^2$$
, $I_2 = 0.5 \text{ kgm}^2$, $I_3 = 0.5 \text{ kgm}^2$,

 $h_1 = 10 \,\mathrm{kgm^2 rad/s}, h_2 = 5 \,\mathrm{kgm^2 rad/s}, h_3 = 17 \,\mathrm{kgm^2 rad/s},$

$$W_1(0) = 0.5 \text{ rad/s}, W_2(0) = 0.1 \text{ rad/s}, W_3(0) = -1 \text{ rad/s},$$

$$\mu_1(0) = 0.3532, \ \mu_2(0) = 0.1466, \ \mu_3(0) = 0.6118$$

 $\hat{\mu}_1(0) = 0.8468, \, \hat{\mu}_2(0) = 0.9534, \, \hat{\mu}_3(0) = -0.2882.$ (34)

The results are shown in Figures 1(a)-3(d). Figures 1(a)-1(d) show the time histories of state variables, error attitude parameters and control moments for the values $C_1 = 0.1$, $C_2 = 0.2$, $C_3 = 0.3$, k = 100, $l_1 = 25$, $l_2 = 30$, $l_3 = 45$. Figures 2(a)-2(d) show the time histories of state variables, error attitude parameters and control moments for the values $C_1 = 0.3$, $C_2 = 0.4$, $C_3 = 0.5$, k = 200, $l_1 = 220$, $l_2 = 240$, $l_3 = 260$, Figures 3(a)-3(d) show the time histories of state variables, error attitude parameters and control moments for the values $C_1 = 0.6$, $C_2 = 0.8$, $C_3 = 0.6$, k = 5000, $l_1 = 500$, $l_2 = 520$, $l_3 = 550$, Recall that increasing the values of the friction coefficients of the rotors system, control con-

stants and stability constants has the effect of decreasing the time for the control process, but these figures have the same behavior.

Based on the above numerical simulation study we conclude that, for the same initial state the control process depends on the friction coefficients of the rotors system, control constants and stability constants. The necessary time for the control process depend upon the friction coefficients, control constants and stability constants and becomes more effective for the large values of these control.

5. Conclusions

In this paper, the control law (21) which stabilizes asymptotically the rotational motion (15) of a rigid body in terms of the MRPs is derived. The inertial frictions of the rotors are taken into account. Global asymptotic stability is shown by applying Matrosov theorem. The equilibrium position (30) of the rigid body which occurs when the principal axes of inertia of the body coincide with the inertial axes is studied to be asymptotically stable as a spe-



Figure 1. Show the above data at $C_1 = 0.1$, $C_2 = 0.2$, $C_3 = 0.3$, k = 100, $l_1 = 25$, $l_2 = 30$, $l_3 = 45$.



Figure 2. Show the above data at $C_1 = 0.3$, $C_2 = 0.4$, $C_3 = 0.5$, k = 200, $l_1 = 220$, $l_2 = 240$, $l_3 = 260$.



Figure 3. Show the above data at $C_1 = 0.6$, $C_2 = 0.8$, $C_3 = 0.6$, k = 5000, $l_1 = 500$, $l_2 = 520$, $l_3 = 550$.

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cial case of the studied problem. Numerical examples of the results are presented.

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